Structural Analysis Formulas:

BENDING:

$$\sigma_z = \frac{\left(M_x I_{yy} - M_y I_{xy}\right)y + \left(M_y I_{xx} - M_x I_{xy}\right)x}{I_{xx}I_{yy} - I_{xy}^2} \quad (2.4)$$
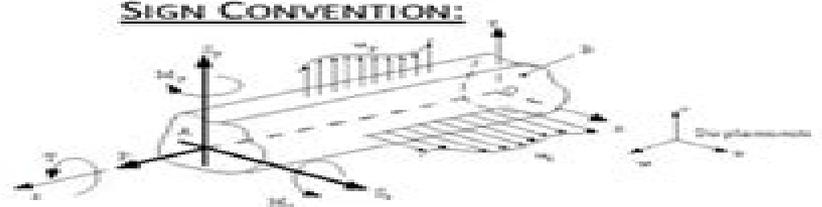
TORSION:

Circular section: $\tau = \dfrac{T\rho}{J}$ (3.1) $\dfrac{d\theta}{dz} = \dfrac{T}{GJ}$ (3.2)

Thin-walled
Closed section: $\tau = \dfrac{q}{t} = \dfrac{T}{2tA_m}$ (3.3) $\dfrac{d\theta}{dz} = \dfrac{T}{4A_m^2 G}\displaystyle\oint_{s} \dfrac{1}{t}ds$ (3.4)

Thin plate: $\tau_{max} = \dfrac{3T}{st^2}$ (3.5) $\dfrac{d\theta}{dz} = \dfrac{3T}{Gst^3}$ (3.6)

SHEAR:

Open section: $q_s = -\dfrac{V_x I_{yy} - V_y I_{xy}}{I_{xx}I_{yy} - I_{xy}^2}\displaystyle\int_0^s t y\,ds - \dfrac{V_y I_{xx} - V_x I_{xy}}{I_{xx}I_{yy} - I_{xy}^2}\displaystyle\int_0^s tx\,ds$ (4.6)

Closed section: $q_s = q_b + q_{s0} = -\dfrac{V_x I_{yy} - V_y I_{xy}}{I_{xx}I_{yy} - I_{xy}^2}\displaystyle\int_0^s t y\,ds - \dfrac{V_y I_{xx} - V_x I_{xy}}{I_{xx}I_{yy} - I_{xy}^2}\displaystyle\int_0^s tx\,ds + q_{s0}$ (4.7)

SECTION PROPERTIES:

Shear centre: (general)   Equate moment due to resultant force with moment due to internal shear flow

Shear centre: (closed section)   In addition to above, require: $q_{s0} = -\dfrac{\oint (q_b/Gt)\,ds}{\oint ds/Gt}$ (4.9)

Moments of Inertia: $I_{xx} = \displaystyle\int_A y^2 dA$ $I_{yy} = \displaystyle\int_A x^2 dA$ $I_{xy} = \displaystyle\int_A xy\,dA$

Parallel Axis Theorem:
$$I_{ii}^A = I_{ii}^B + Ac_j^2$$
$$I_{ij}^A = I_{ij}^B + Ac_i c_j$$
$$i = x, y, \text{ or } z$$
(2.9)

Solid Rectangular Section   Solid Circular Section   Thin-walled Circular Section

$I_{xx} = \dfrac{ta^3 \sin^2 \beta}{12}$ (2.6)

$I_{yy} = \dfrac{ta^3 \cos^2 \beta}{12}$ (2.7)

$I_{xy} = \dfrac{ta^3 \sin\beta\cos\beta}{12}$ (2.8)

$I_x = \dfrac{hb^3}{12}$

$I_x = \dfrac{\pi d^4}{64}$ $J = \dfrac{\pi d^4}{32}$

$I_x = \dfrac{\pi r d^3}{8}$ $J = \dfrac{\pi r d^3}{4}$

$I_{xx} = \displaystyle\sum_{r=1}^{n} y_r^2 B_r$ (6.1) $I_{yy} = \displaystyle\sum_{r=1}^{n} x_r^2 B_r$ (6.2) $I_{xy} = \displaystyle\sum_{r=1}^{n} x_r y_r B_r$ (6.3)

IDEALIZED STRUCTURES:

Skin contribution to boom

$B_1 = \dfrac{t_D b}{6}\left(2 + \dfrac{\sigma_2}{\sigma_1}\right)$ (6.5)

$B_2 = \dfrac{t_D b}{6}\left(2 + \dfrac{\sigma_1}{\sigma_2}\right)$ (6.6)

Zero thickness skins & point area booms

SIGN CONVENTION:

Note: use of convention that + moments create + stress in + quadrant!
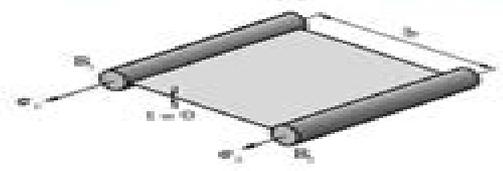
Structural Analysis V 1

Emilie Sanchez

Structural Analysis V 1:

Unveiling the Energy of Verbal Artistry: An Psychological Sojourn through **Structural Analysis V 1**

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